

Transverse subharmonic vibration of an axially-excited beam

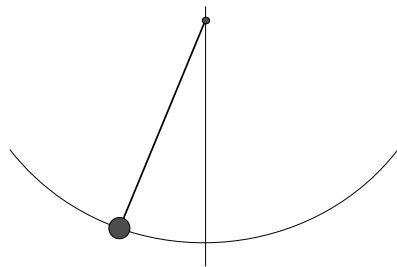
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Definition of the argumentary oscillator.



Vertical
coil

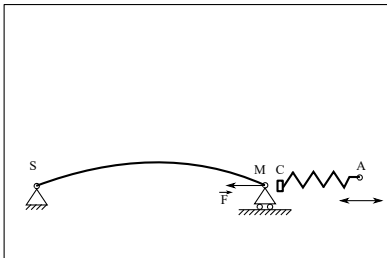
- History: Béthenod, Penner, Doubochinski.
- Oscillator often of Duffing type.
- Submitted to an external periodic force.
- Modulation-coupling by spatial position.
- Equation (reduced time):

$$\frac{d^2 x}{dt^2} + 2\beta \frac{dx}{dt} + x + \mu x^3 = A H(x) F(t)$$

with $F(t)$ =periodic function of time.

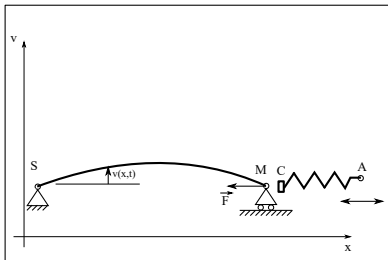
- Modelling to find the H-function.

Setup.



- Beam:
 - Articulated on the left (point S)
 - Articulated-guided on the right (point M)
- Point A:
 - Forced harmonic horizontal movement
 - Linked to C by a linear spring
 - C in partial contact with M

PDE.



Equation of the beam's motion:

$$\rho S \ddot{v} + EI v^{(4)} - F v^{(2)} + 2\beta \dot{v} = f$$

where

- x = longitudinal coordinate
- $v(x, t)$ = transverse coordinate
- ρ = density,
- S = section area,
- e = Yung's modulus,
- I = quadratic moment,
- $F(t)$ = axial force applied to M,
- β = damping,
- $f(x, t)$ = transverse force's linear density.

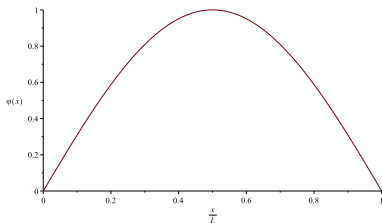
F force.

$$F(x_M, t) = F_0 + k(l(t) - l_0)$$

where

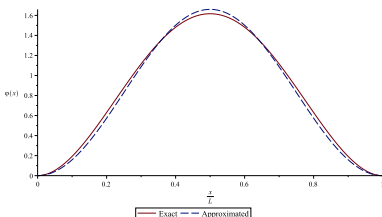
- $l(t)$ = actual spring length
- l_0 = idle spring length
- k = spring stiffness
- F_0 = force applied to the beam when beam is in rectilinear position and point A at center position

First-mode shapes.



Case articulated-(articulated-guided):

$$q_1(x) = \sin\left(\frac{\pi x}{L}\right)$$

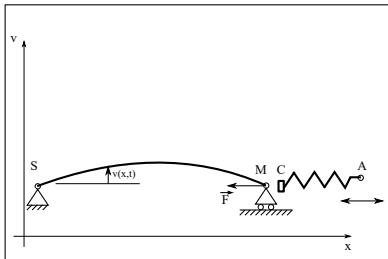


Case clamped-(clamped-guided):

$$q_1(x) \approx b \left(1 - \cos\left(\frac{2\pi x}{L}\right)\right)$$

with $b = 0.83$

M's abscissa.



Inextensible beam $\rightarrow s(x_M, v, t) = L :$

$$s(x_M, v, t) = \int_0^{x_M} \sqrt{1 + \left(\frac{\partial v(u, t)}{\partial u}\right)^2} du = L$$

$$\Rightarrow x_M(t) \approx L - \frac{1}{2} \int_0^L \left(\frac{\partial v(u, t)}{\partial u}\right)^2 du$$

Case articulated-(articulated-guided), first mode :

$$v(x, t) = Lq_1(t) \sin\left(\pi \frac{x}{L}\right) \Rightarrow x_M(t) \approx L \left(1 - \frac{\pi^2}{4} q_1^2(t)\right)$$

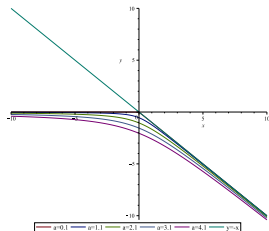
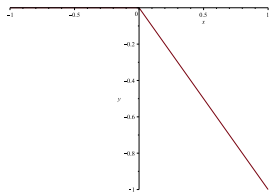
Case clamped-(clamped-guided), first mode :

$$v(x, t) \approx b \left(1 - \cos\left(\frac{2\pi x}{L}\right)\right) \Rightarrow x_M(t) \approx L \left(1 - \pi^2 q_1^2(t)\right)$$

Contact law.

Principle: soft transition between contact and non-contact, by smoothing the curve of $y(x) = -\frac{x + |x|}{2}$.

Purpose: avoid solver's discrepancies.



$$y = -\frac{x + |x|}{2} \text{ a } C^0 \text{ function.}$$

$$y = -\frac{x + \sqrt{x^2 + a^2}}{2}, \text{ a } C^\infty \text{ function.}$$

$a = 0.1 \dots 4.1$ in figure.

Modal projection.

For the p-th modal coordinate, case articulated-(articulated-guided):

$$\ddot{q}_p(t) + 2 \frac{\beta}{\rho S} \dot{q}_p(t) + \left(\frac{p\pi}{L}\right)^4 \frac{EI}{\rho S} q_p(t) + \left(\frac{p\pi}{L}\right)^2 \frac{EI}{\rho S} F(q_1, t) q_p(t) - \frac{2}{L^2 \rho S} \int_0^L \sin\left(\frac{p\pi}{L} x\right) f(x, t) dx = 0$$

For the first mode, case articulated-(articulated-guided):

$$\ddot{q}_1(t) + 2 \frac{\beta}{\rho S} \dot{q}_1(t) + \left(\frac{\pi}{L}\right)^4 \frac{EI}{\rho S} q_1(t) + \left(\frac{\pi}{L}\right)^2 \frac{EI}{\rho S} F(q_1, t) q_1(t) - \frac{2}{L^2 \rho S} \int_0^L \sin\left(\frac{\pi}{L} x\right) f(x, t) dx = 0$$

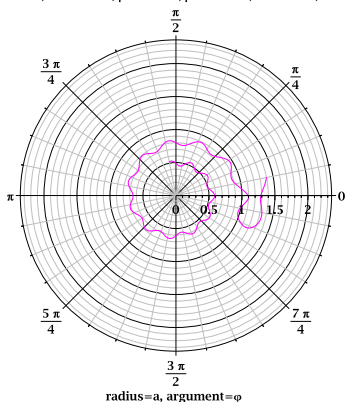
For the first mode, case clamped-(clamped-guided):

$$\ddot{q}_1(t) + 2 \frac{\beta}{\rho S} \dot{q}_1(t) + \left(\frac{2\pi}{L}\right)^4 \frac{EI}{3\rho S} q_1(t) + \left(\frac{2\pi}{L}\right)^2 \frac{EI}{3\rho S} F(q_1, t) q_1(t) - \frac{2}{3L^2 \rho S} \int_0^L \left(1 - \cos\frac{2\pi}{L} x\right) f(x, t) dx = 0$$

In the simulations, we assume $f(x, t) \equiv 0$.

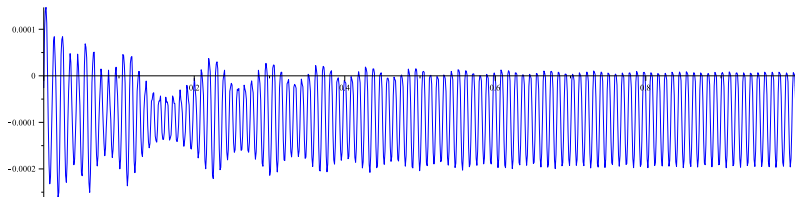
Van der Pol amplitude/phase representation.

$v = 63.0, \lambda = 10.0357, \beta = 0.0050, \mu = -0.1667, A = 2.100, n = 11.$

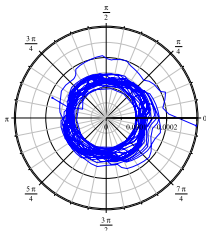


- The averaging method searches an approximate solution $x = a(t) \sin(\omega t + \varphi(t))$, where $a(t)$ and $\varphi(t)$ are slowly-varying functions of time.
- $(a, \varphi) \rightarrow$ Natural representation: van der Pol.
- Example of integral curve winding up around the origin.
- For the simulations: $n = \frac{\text{Excitation frequency}}{\text{Oscillator frequency}} = 4$ to 10 (even integer), $\beta = 0.0001 \dots 0.01$.
- Polar form is better, due to invariance vs $\frac{2\pi}{n}$ rotation. Radius=a, argument= φ .

Preliminary simulation results: result #1.



Equation 2nd ordre d'origine, extraction via x et x_point, a0=0.0003, q0=-0.0873009182, decalage_phi0=-0.8, A=0.0004, ph_offset=0, n=4, nu_excitation=1990.041208, omega_final_oscillateur_estime=497.5103019, periode=0.01262925669, p=0.01, mu=10000.00000, r_max=1, nb_pmts=1000



Simulation duration: 1 s

Buckling force: 6900 N

Force in point A: 3 N max

$$f_0 = 80 \text{ Hz}, \quad \frac{\nu}{\omega} = 4, \quad \frac{\omega}{\omega_0} = 1.01$$

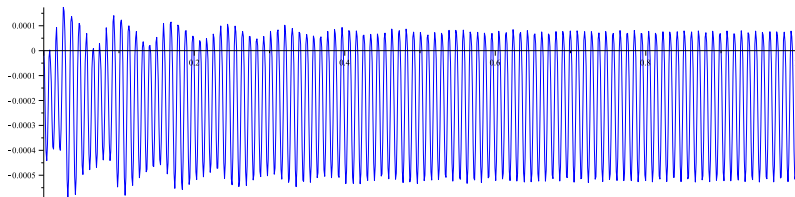
$$E = 70 \cdot 10^9 \text{ Pa}, \quad \rho = 2700 \text{ kg/m}^3, \quad I = 10^{-8} \text{ m}^4,$$

$$S = 10^{-4} \text{ m}^2, \quad L = 1 \text{ m}, \quad \beta = 10^{-2},$$

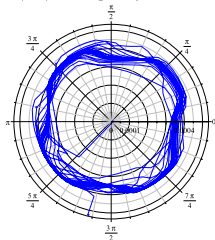
$$\text{Point A's amplitude} = 4 \cdot 10^{-7} \text{ m},$$

$$F_0 = 0.0 \text{ N}.$$

Preliminary simulation results: result #2.



Equation 2nd ordre d'origine, extraction via x et x_point, a0=0.00001, phi=0, decalage, phi0=0, A=0.0004, phi_offset=0, n=4, nu excitation=1989.643159, omega final oscillateur estimee=497.4107898, periode1=0.01263178330, beta=0.01, mu=10004.00160, r_max=1, nb pats=1000, x0=0, x1=0.004974107898



Simulation duration: 1 s

Buckling force: 6900 N

Force in point A: 5.6 N max

$$f_0 = 80 \text{ Hz}, \quad \frac{\nu}{\omega} = 4, \quad \frac{\omega}{\omega_0} = 1.01$$

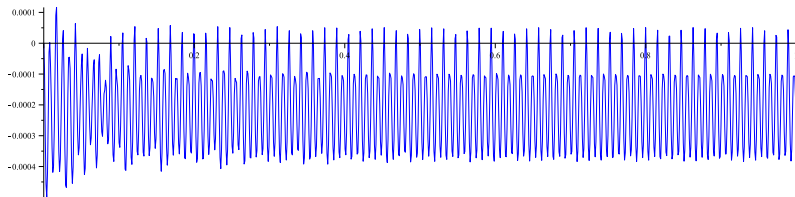
$$E = 70 \cdot 10^9 \text{ Pa}, \quad \rho = 2700 \text{ kg/m}^3, \quad I = 10^{-8} \text{ m}^4,$$

$$S = 10^{-4} \text{ m}^2, \quad L = 1 \text{ m}, \quad \beta = 10^{-2},$$

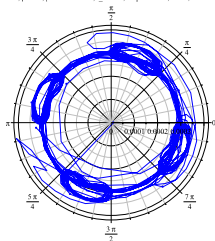
$$\text{Point A's amplitude} = 4 \cdot 10^{-7} \text{ m},$$

$$F_0 = -2.7 \text{ N}.$$

Preliminary simulation results: result #3.



Equation 2nd ordre d'origine, extraction via x et x_point, a0=0.00001, phi=0, decalage, phi0=0, A=0.0004, phi_offset=0, n=4, nu excitation=1989.543636, omega final oscillateur estimée=497.3859091, periode1=0.01263241518, beta=0.01, mu=10005.00250, t_max=1, nb pats=1000, x0=0, x1=0.004973859090



Simulation duration: 1 s

Buckling force: 6900 N

Force in point A: 6.2 N max

$$f_0 = 80 \text{ Hz}, \quad \frac{\nu}{\omega} = 4, \quad \frac{\omega}{\omega_0} = 1.01$$

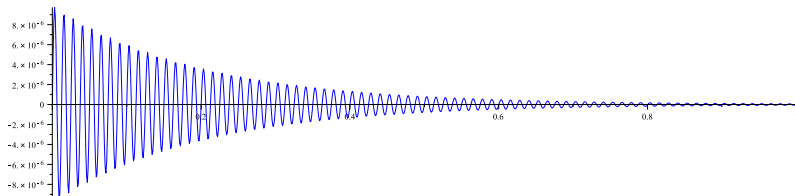
$$E = 70 \cdot 10^9 \text{ Pa}, \quad \rho = 2700 \text{ kg/m}^3, \quad I = 10^{-8} \text{ m}^4,$$

$$S = 10^{-4} \text{ m}^2, \quad L = 1 \text{ m}, \quad \beta = 10^{-2},$$

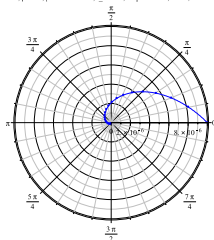
$$\text{Point A's amplitude} = 4 \cdot 10^{-7} \text{ m},$$

$$F_0 = -3.5 \text{ N}.$$

Preliminary simulation results: result #4.



Equation 2nd ordre d'origine, extraction via x et x_point, a0=0.00001, q0=0, decalage_phi0=0, A=0, ph_offset=0, n=4, nu_excitation=2089.543268, omega_final_oscillateur_estime=497.5103019, periode=0.01262925669, beta=0.01, mu=10000.00000, t_max=1, nb_pots=1000, x0=0, x1=0.004975103019



Simulation duration: 1 s

Buckling force: 6900 N

Force in point A: 0 N max

$$f_0 = 80 \text{ Hz}, \quad \frac{\nu}{\omega} = 4, \quad \frac{\omega}{\omega_0} = 1.01$$

$$E = 70 \cdot 10^9 \text{ Pa}, \quad \rho = 2700 \text{ kg/m}^3, \quad I = 10^{-8} \text{ m}^4,$$

$$S = 10^{-4} \text{ m}^2, \quad L = 1 \text{ m}, \quad \beta = 10^{-2},$$

Point A's amplitude = 0 m,

$F_0 = 0 \text{ N}$.

Future work

- Simulations based on complete original motion PDE.
- Other contact conditions.
- Experimentation.

Thank you for your attention.